MODELING OF RAW CANE-SUGAR SYRUP CONCENTRATION USING DIRECT CONTACT MEMBRANE DISTILLATION

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Membrane distillation is an emerging technology for separations that are traditionally accomplished by conventional separation processes such as distillation and reverse osmosis. The membrane distillation driving force is the transmembrane vapor pressure difference that may be maintained with an aqueous solution colder than the feed solution in direct contact with the permeate side of the membrane giving rise to the configuration known as direct contact membrane distillation (DCMD). This process is ideally suited for the concentration of aqueous streams such as fruit juice and sugar solutions.

The present numerical study is carried out to describe how the technique of DCMD can be applied to the concentration of cane-sugar syrup. The main objective of the present study is to provide a detailed numerical analysis of the heat and mass transfer in DCMD and to offer useful basic detailed information about the nature of the process that is needed for process improvement and optimization. In this regards, the present study is carried out to explore the effects of parameters such as the feed temperature, the feed concentration and the hydrodynamics of the hot and cold solutions on the distillate volume flow.

The developed method allows solving numerically the hydrodynamic, heat and mass transport equations with permeation taken into account. Velocity and temperature distributions inside the membrane feed and cold solution channels were obtained, as well as the concentration profiles of the cane-sugar syrup in the membrane feed channel. Some of the principal conclusions drawn from the present study are: (1) the distillate volume flux increases with the feed temperature, (2) the distillate volume flow decreases as the feed initial concentration increases, and (3) the distillate volume flow increases with the flow rate through the feed channel. The results were compared with the available data and the agreement is satisfactory.

KEYWORDS: Membrane distillation; Direct contact membrane distillation; Modeling; Cane-sugar syrup, Concentration.

NOMENCLATURE			
Alphabetic Symbols c concentration (kg /kg) c_p specific heat capacity (kJ/kg K) D_s solute diffusion coefficient (m²/s) H channel height (m) J_v mass flux (kg/m².s) k_s solute thermal conductivity (W/m K) L channel length (m) p pressure (Pa) S_{ϕ} source term of the variable ϕ S_{ϕ} source term of the variable ϕ	yCartesian coordinate normal to the membraneGreek Symbols δ membrane thickness (m) ε membrane porosity (%) ϕ any of variables to be solved Γ_{ϕ} diffusion coefficient of variable ϕ μ dynamic viscosity (Pa.s) v kinematics viscosity = μ/ρ (m²/s) ρ density (kg/m³)		
$ \begin{array}{ll} S_p & \text{coefficient in the discretized source} \\ & \text{term} \\ S_u & \text{coefficient in the discretized source} \\ & \text{term} \\ T & \text{temperature (°C)} \\ u & \text{streamwise velocity (m/s)} \\ v & \text{velocity in } y \text{ direction (m/s)} \\ x & \text{Cartesian coordinate in the} \\ & \text{streamwise direction} \end{array} $	$ \begin{array}{ll} \tau & \text{membrane tortuosity} \\ \hline Subscripts \\ b & \text{bulk} \\ h & \text{hot stream} \\ i & \text{inlet} \\ c & \text{cold stream} \\ m & \text{membrane} \\ s & \text{solute, or solid} \\ \end{array} $		

1. INTRODUCTION

Membrane distillation (MD) is an emerging technology for separations that are traditionally accomplished by conventional separation processes such as distillation or reverse osmosis. Since its appearance, MD claims to be a cost effective separation process that can utilize low-grade waste and/or alternative energy sources such as solar and geothermal energies. The potential advantages of MD process in comparison to the conventional separation processes rely on the lower operating temperature and hydrostatic pressure. Feed solutions having temperatures much lower than its boiling point under pressures near atmosphere can be used.

Direct contact membrane distillation (DCMD), in which two solutions at different temperatures are in direct contact with the membrane surfaces, appears as the best configuration for applications in which the major feed component is water, such as desalination and concentration of aqueous solutions. DCMD has found success in areas where treatments at lower temperatures are preferred over that at higher temperatures in order to safeguard product quality [1,2]. DCMD was also investigated for desalination applications. Some studies [3,4] were carried out to produce fresh water with a rejection factor of 100%.

In a conventional cane sugar manufacturing process [5], the clarified juice (17- 20° Brix) which comes as an overflow from the Dorr at a temperature of about 100°C proceeds either to a rising film evaporator (Kestner) or a falling film evaporator (calandria) of the vertical /horizontal type. The falling film evaporator (calandria) is operated with steam (135°C, and 2.1 kg/cm²). From here the juice goes to a multi-

effect evaporator. The consumption of steam for the removal of water from this juice is considerable. If one were to put a membrane distillation step before the juice goes to the evaporators, it is technically feasible to increase the solids in juice from 17° to 30° Brix. The numerical study was carried out to simulate this step.

As an attractive separation process, MD has been the subject of worldwide academic studies by many experimentalist and theoreticians [e.g. 1-4]. Recently, Sanjay Nene *et al.* [6] applied DCMD for the concentration of raw cane-sugar syrup. They pointed out that it's technically feasible to introduce a membrane distillation step to increase the solids in juice from 17 to 30° Brix. The aim of their experiments was to study the flux decay in membrane distillation when the run conditions were similar to those in the sugar industry. The obtained results indicate that it is possible to continuously remove water from the cane sugar solution at steady-state value with the capacity of about 10.0 kg/m² h. They also claimed that MD saves energy in removing water from clarified cane juice by utilizing sensible heat.

Computational fluid dynamics (CFD), together with mass transfer modeling, has been proved to be a powerful tool to be used in the feed and permeate sides of membrane modules to effect the predictions of velocities, pressure, temperature and solute concentration, variables that are crucial for the management of the syrup concentration process. The present study pertains to modeling numerically of a canesugar syrup concentration using DCMD. The study is directed towards establishing a numerical method capable of predicting the flow and concentration characteristics of the DCMD taking into account the variable properties of the streams. For this purpose, a finite volume discrete scheme using the SIMPLE (Semi-Implicit Method for Pressure Linked Equations) pressure-correction scheme combined with QUICK (Quadratic Upwind Interpolation Convective Kinematics) scheme in the frame of staggered grid is used.

The present numerical study is carried out to describe how the technique of DCMD can be applied to the concentration of cane-sugar syrup. The main objective of the present study is to provide a detailed numerical analysis of the heat and mass transfer in DCMD and to offer useful basic detailed information about the nature of the process that is needed for process improvement and optimization. In this regards, the present study is carried out to explore the effects of parameters such as the feed temperature, the feed concentration and the hydrodynamics of the hot and cold solutions on the distillate volume flow.

2. MODEL DEVELOPMENT

The system to be studied consists of a porous hydrophobic membrane, which is held between two symmetric channels shown in Fig. 1. The configuration is similar to a heat exchanger working in counter-current flow with the warm cane-sugar syrup solution flowing in the lower channel and the cold water solution on the upper side of the membrane module. Mass and heat balance equations were written to describe the concentration and temperature profiles on both the hot and cold sides. The hot and cold fluids counter-flow tangentially to the membrane surface in a flat membrane module. The temperature difference through the membrane gives rise to a water vapor pressure difference and, consequently to a water flux, *J*, through the membrane. The process is modeled by solving the 2-dimentional momentum, energy and species equations in the hot and cold solutions regions.

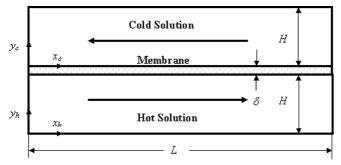


Fig. 1 Membrane module including the membrane and the hot and cold solutions channels

2.1. The Hot Solution Region

The hot syrup solution flows between two parallel walls, the upper wall is hydrophobic microporous membrane and the lower one is impermeable wall. The transport steady state, two dimensional transports of the momentum, energy, and species of the hot solution are described by the continuity, momentum, energy and species conservation equations [7];

$$\frac{\partial u_h}{\partial x_h} + \frac{\partial v_h}{\partial y_h} = 0 \tag{1}$$

$$\frac{\partial(\rho_s u_h u_h)}{\partial x_h} + \frac{\partial(\rho_s v_h u_h)}{\partial y_h} - \frac{\partial}{\partial x_h} [\mu_s(\frac{\partial u_h}{\partial x_h})] - \frac{\partial}{\partial y_h} [\mu_s(\frac{\partial u_h}{\partial y_h})] + \frac{\partial P_h}{\partial x_h} = 0$$
(2)

$$\frac{\partial(\rho_s u_h v_h)}{\partial x_h} + \frac{\partial(\rho_s v_h v_h)}{\partial y_h} - \frac{\partial}{\partial x_h} [\mu_s(\frac{\partial v_h}{\partial x_h})] - \frac{\partial}{\partial y_h} [\mu_s(\frac{\partial v_h}{\partial y_h})] + \frac{\partial P_h}{\partial y_h} = 0$$
(3)

Where, u_h and v_h are the streamwise and transverse velocity components of the hot solution, respectively, *P* is the pressure, ρ is the density and μ is the dynamic viscosity.

Mass transfer occurring within domains with porous walls can be mathematically expressed by the two dimensional convective and diffusion equation as follows [8];

$$\frac{\partial(\rho_s u_h c_s)}{\partial x_h} + \frac{\partial(\rho_s v_h c_s)}{\partial y_h} - \frac{\partial}{\partial x_h} [D_s(\frac{\partial c_s}{\partial x_h})] - \frac{\partial}{\partial y_h} [D_s(\frac{\partial c_s}{\partial y_h})] = 0$$
(4)

Where D_s is the solute diffusion coefficient and c_s is the solute concentration.

Most of the previous models solve equations (1)-(4) using constant or concentration-dependent only thermophysical and flow properties. In the present study, the energy equation has to be solved to account for the temperature dependence of the abovementioned properties. For two dimensional, incompressible, steady laminar channel flow; the energy equation is given as [8]

$$\frac{\partial(\rho_s u_h T_h)}{\partial x_h} + \frac{\partial(\rho_s v_h T_h)}{\partial y_h} - \frac{\partial}{\partial x_h} \left[\frac{k_s}{c_{ps}} \left(\frac{\partial T_h}{\partial x_h}\right)\right] - \frac{\partial}{\partial y_h} \left[\frac{k_s}{c_{ps}} \left(\frac{\partial T_h}{\partial y_h}\right)\right] = 0$$
(5)

where k_s and c_{ps} are the hot solution thermal conductivity and specific heat at constant pressure, respectively.

2.2. The Membrane Domain

In DCMD the pressure difference at two sides of the membrane will be zero when both the feed and permeate flows are under atmospheric pressure. In this case the contribution of Poiseuille flow to mass transfer can be neglected. At the typical membrane temperature of 60 °C, the mean free path of water vapor is 0.11 μ m and the mean pore diameter of membranes is 0.1 - 0.5 μ m. Therefore, in the system where water (component A) is used as volatile component Kn (Knudsen number) may vary from 0.2 to 1.0. As air (component B) is trapped in the pores, the permeation of water vapor through the membrane is regulated by the Knudsen-molecule diffusion transition mechanism.

For mass transfer through the membrane in DCMD, on the ground of Knudsen-molecule diffusion transition model, the following equation can be obtained to calculate trans-membrane mass flux, J_{ν} [9]:

$$J_{\nu}(x) = \frac{\varepsilon M_{A}}{\tau \delta} \frac{p D_{AB}}{RT_{m}} \ln \frac{(p - p_{pm})/p D_{AB} + (3/4d) \sqrt{2\pi M_{A}/RT_{m}}}{(p - p_{fm})/p D_{AB} + (3/4d) \sqrt{2\pi M_{A}/RT_{m}}}$$
(6)

where ε (porosity), δ (thickness), d (pore diameter) and τ (tortuosity), are membrane geometry parameters, T_m is the hot and cold streams average temperature (K), R is the universal gas constant, M_A is the water vapor molecular weight, p_{pm} and p_{fm} are the vapor partial pressures (Pa) at the membrane-permeate and feed interfaces respectively calculated by using [4]:

$$p^{\nu} = \exp\left(23.328 - \frac{3841}{T - 45}\right) \tag{7}$$

p the total (air + vapor) pressure, and D_{AB} is the diffusion coefficient of the vapor through the air (m²/s) at the temperature of 273 - 373 K which is estimated from the following empirical equation [4]:

$$pD_{AB} = 1.895 \text{ x } 10^{-5} \text{ T}^{2.072}$$
(8)

where the unit of pD_{AB} is Pa-m²/s.

On the other hand, the resistances in the heat transfer process of DCMD consist of three parts: the resistance of boundary layer at the feed side, of the membrane and of the boundary layer at the permeate side. The total heat flux is transferred from the hot surface of the membrane to the cold surface of the membrane by two different parallel routes. One is by heat conduction across the membrane material (Q_c) while the other is by the mass transfer of the vapor (Q_L)

$$Q_{T} = Q_{L} + Q_{C} = J_{v} \Delta H_{v} + (k_{m} / \delta)(T_{fm} - T_{pm})$$
(9)

where ΔH_v is the latent heat of vaporization. In Eq. (9) k_m is the thermal conductivity of the porous membrane that can be calculated as

$$k_m = \varepsilon k_g + (1 - \varepsilon) k_s$$

with k_g and k_s being the thermal conductivities of the gas and solid phases [10].

2.3. The Cold Solution Region

The cold solution flows between two parallel walls, the lower wall is hydrophobic microporous membrane and the upper one is impermeable wall. The transport of the momentum and energy of the cold solution are described by the continuity, momentum, and energy conservation equations [7];

$$\frac{\partial u_c}{\partial x_c} + \frac{\partial v_c}{\partial y_c} = 0 \tag{10}$$

$$\frac{\partial(\rho_w u_c u_c)}{\partial x_c} + \frac{\partial(\rho_w v_c u_c)}{\partial y_c} - \frac{\partial}{\partial x_c} [\mu_w (\frac{\partial u_c}{\partial x_c})] - \frac{\partial}{\partial y_c} [\mu_w (\frac{\partial u_c}{\partial y_c})] + \frac{\partial P_c}{\partial x_c} = 0 \quad (11)$$

$$\frac{\partial(\rho_w u_c v_c)}{\partial x_c} + \frac{\partial(\rho_w v_c v_c)}{\partial y_c} - \frac{\partial}{\partial x_c} [\mu_w (\frac{\partial v_c}{\partial x_c})] - \frac{\partial}{\partial y_c} [\mu_w (\frac{\partial v_c}{\partial y_c})] + \frac{\partial P_c}{\partial y_c} = 0$$
(12)

Where, u_c and v_c are the streamwise and transverse velocity components of cold stream, respectively, *P* is the pressure, ρ is the density and μ is the dynamic viscosity.

In the present study, the energy equation has to be solved to account for the temperature dependence of the cold stream properties. For two dimensional, incompressible, steady laminar channel flow; the energy equation is given as [8]

$$\frac{\partial(\rho_w u_c T_c)}{\partial x_c} + \frac{\partial(\rho_w v_c T_c)}{\partial y_c} - \frac{\partial}{\partial x_c} \left[\frac{k_w}{c_{pw}} \left(\frac{\partial T_c}{\partial x_c}\right)\right] - \frac{\partial}{\partial y_c} \left[\frac{k_w}{c_{pw}} \left(\frac{\partial T_c}{\partial y_c}\right)\right] = 0$$
(13)

where k_w and c_{pw} are the cold solution thermal conductivity and specific heat at constant pressure, respectively.

2.4 Boundary Conditions

The boundary conditions of the problem are specified as the followings:

- 1- At the inlet of both channels, the flow is assumed to be fully developed thus a parabolic flow is specified. A uniform inflow concentration of c_{sb} is specified at the inlet of the hot solution channel. A constant inlet temperature of T_{hb} and T_{cb} are specified at the inlet of the hot and cold channels, respectively.
- 2- At the lower wall of the hot channel, the tangential velocity u and the transverse velocity v are set to zero; the normal gradients of the concentration and temperature are set to zero.

- 3- At the upper wall of the cold channel, the tangential velocity u and the transverse velocity v are set to zero; the normal gradients of the temperature are set to zero.
- 4- At the membrane walls, the conditions are more complex, as flow permeates through the wall. The tangential velocity u is set to zero i.e. no slip at membrane walls. Variation in permeation was modeled using the following expression;

 $v_{mh}(x) = J_v(x)/\rho_s$

The boundary condition of the concentration at the membrane lower wall results from a balance of the convective and diffusive fluxes. The concentration boundary condition is given by;

$$\left. D_s \frac{\partial c_s(x)}{\partial y} \right|_{y=H} = J_v(x) / \rho_s$$

The boundary condition of the temperature at the membrane walls results from a balance of the convective and conductive heat fluxes. The temperature boundary condition is given by;

$$-k_s \frac{\partial T(x)}{\partial y} \bigg|_{y=0 \text{ or } H} = Q_C(x) + J_v(x) \Delta H_v$$

5- At the exit, both the flow, temperature and concentration fields are assumed to obey the boundary layer approximation. It is important to mention that this treatment of the down stream end boundary condition has proved to be robust and effective in shortening the computational domain leading to the reduction of the number of grid nodes [11].

2.5 Discretization

The steady-state form of the conservation equations of continuity, momentum, concentration and energy can be written in a general form as [12];

$$\frac{\partial(\rho u\phi)}{\partial x} + \frac{\partial(\rho v\phi)}{\partial y} - \frac{\partial}{\partial x} [\Gamma_{\phi}(\frac{\partial\phi}{\partial x})] - \frac{\partial}{\partial y} [\Gamma_{\phi}(\frac{\partial\phi}{\partial y})] - S_{\phi} = 0$$
(14)

where ϕ stands for any of the variables to be solved, Γ_{ϕ} is the diffusion coefficient, and S_{ϕ} is the source term of the variable ϕ . For $\phi = u$ or v and $\Gamma_{\phi} = \mu$ one gets the momentum equations, while for $\phi = 1$ and $\Gamma_{\phi} = 0$ one obtains the continuity equation [13]. If $\phi = T$ and $\Gamma_{\phi} = k_{f}/c_{p}$ one gets the energy equation. When $\phi = c$ and $\Gamma_{\phi} = D$, the general equation stands for the mass transfer equation [14].

All the governing equations are discretized by first integrating them over a control volume (CV) and then approximating the fluxes of variable crossing the faces of each cell in terms of the values at the neighboring grid points. In the present work, a QUICK scheme, which can handle uniform and non-uniform grid systems, is used to

finite differencing the convective terms and to secure second order accuracy in central differencing the diffusive fluxes. The resulting finite-difference equations are described in the form of [15],

$$a_p \phi_p = \sum_i a_i \phi_i + S_u \Delta V , \qquad i = E, W, N, S, EE, WW, NN, SS, \qquad (15a)$$

$$a_p = \sum_i a_i - S_p \Delta V, \qquad i = E, W, N, S, EE, WW, NN, SS, \qquad (15b)$$

where ΔV is the cell volume and S_p and S_u are the coefficients appearing in the following linearized source term;

$$S_{\phi} = S_u + S_p \phi \tag{15c}$$

The finite difference coefficients a_i are the coefficients describing the magnitudes of the sum of the convective and diffusive fluxes and contain the geometric properties of the control volume [11].

2.6 Numerical Procedure

The present study utilizes a modified version of the SIMPLE procedures developed by Partaker and Spalding [15]. The main steps of the SIMPLE algorithm are;

- 1- A pressure field is assumed in the hot channel,
- 2- It is used to obtain approximate velocity field in the hot channel,
- 3- The velocity and pressure fields (in the hot channel) are corrected if the former does not satisfy the continuity equation,
- 4- Solve the discretization equations for the other ϕ 's such as temperature and concentration (in the hot channel) provided their influence on the flow field.
- 5- Do step 1-4 in the cold channel. Calculate the membrane permeation and heat transfer through the membrane.
- 6- Return to step 2 with the corrected velocity field and the new values of all other ϕ 's and then the steps 2-5 are repeated until a converged solution is obtained.

In the present work, the cross-stream distribution of *u*-velocity component is adjusted to satisfy the overall continuity (conservation of the mass flow are integrated over a cross-stream line) whereas the pressure field is adjusted to satisfy the overall momentum balance. This procedure is important especially for the present problem in which the flow is changed as the flow moves downstream due to the permeation through the membrane surface [11, 13]. Moreover, the cross-stream distribution of the concentration (in the hot channel) is adjusted to satisfy the overall mass concentration of the permeated species.

An alternating direction implicit (ADI) procedure has been combined with the iterative solution procedure of equations (15) to enhance isotropic propagation of a change of variables occurring at one point to the surrounding [13]. This procedure makes use of the line-by-line TDMA solver. In the ADI procedure, sweep of line- by-line integration was carried out along both north-south grid lines and along east-west grid lines alternatively. The same procedure was applied twice for the pressure correction [11].

2.7 Physical Properties

The membrane solute rejection leads to the development of a solute concentration profile in the hot fluid phase adjacent to the membrane. The hot stream was a pure sucrose solution with a concentration corresponding to the raw cane juice. Expressions for the variation of physical properties with temperature and concentration for sugar solution were taken from sugar technologist manual [16]. The cold stream was liquid water subjected to temperature variations. Therefore, the transport and physical properties of the solutions, in the transport equations (1)–(13), should include the variation with the solute concentration (in the hot stream) and the solution temperature in both cold and hot streams. The correlations relative to the variation with the temperature of the physical and transport properties of the liquid water were collected from different sources [e.g. 17].

3. PROCESS PARAMETERS

The parameters to be evaluated in this work include the averaged permeate flux, the conductive heat transfer, the total heat transfer, the process thermal efficiency.

The averaged permeate flux is obtained by integrating Eq. (6) over the length of the membrane and dividing by the membrane length (L)

$$J = \frac{1}{L} \int_{0}^{L} J_{\nu}(x) \, dx \tag{16}$$

by integrating the second term of Eq. (9), the x-averaged conduction heat flux is

$$Q_{C} = \frac{1}{L} \int_{0}^{L} Q_{C}(x) dx$$
(17)

by integrating the first term of Eq. (9), the x-averaged latent heat flux is

$$Q_{L} = \frac{1}{L} \int_{0}^{L} Q_{L}(x) dx$$
(18)

and by integrating Eq. (9), the total heat transfer is

$$Q_{T} = \frac{1}{L} \int_{0}^{L} Q_{T}(x) dx$$
(19)

The process thermal efficiency can be defined as

$$\eta_{th} = \frac{Q_L}{Q_T} \tag{20}$$

where the numerator is the heat used for the production of the distillate.

Polarization phenomena can be described using a temperature polarization coefficient (TPC) and a concentration polarization coefficient (CPC), defined as [1]:

$$TPC = \frac{1}{L} \int_0^L \frac{T_{mh}(x) - T_{mc}(x)}{T_{bh}(x) - T_{bc}(x)} , \qquad (21)$$

$$CPC = \frac{1}{L} \int_0^L \frac{c_{mh}(x) - c_{mc}(x)}{c_{bh}(x) - c_{bc}(x)} = \frac{1}{L} \int_0^L \frac{c_{mh}(x)}{c_{bh}(x)}$$
(22)

where 'm' and 'b' indicate the values of temperature and concentration on the membrane surface and in the bulk condition, respectively.

4. RESULTS AND DISCUSSION

4.1 Model Validation

The computations are carried out in the rectangular domain shown in Fig. 1. The main results were obtained for computational domain of $H = 0.45 \times 10^{-3}$ m, L = 0.1 m. The membrane employed in this study has the characteristics of 0.8 porosity (ε), a nominal pore size (d_p) of 0.2 µm, thermal conductivity (k_m) of 0.055 W/m K, and thickness (δ_m) of 0.6×10⁻⁴ m. A grid-dependence analysis of the method of solution was performed. The number of elements is chosen to be 16,000 (400x40) because further refinement of the mesh to 24,000 elements produced just a 0.06% difference in J.

Moreover, to ensure that the numerical solution is not affected adversely by the specification of the inlet conditions to the hot and cold liquid flow channels, the sensitivity of the solution to the location where the inlet boundary conditions were specified was investigated. Comparing the velocity distributions for specifying the inlet at a distance of x/H = 100 upstream of the channels' inlets (Fig. 1), and for specifying them at the inlet itself (x/H = 0), has shown no discernible differences.

On the light of the objective of the present work, the validity of the present numerical simulation has been verified by comparing results of the present study with the existing experimental data of Martinez-Diez and Vazquez-Gonzalez [18]. Figure 2 shows the average permeate fluxes predicted by the present model along with the experimental data of Martinez-Diez and Vazquez-Gonzalez [18]. In Fig. 2 the distillate fluxes for the three recirculation rates studied (feed and permeate average velocities of 25, 39, and 53 m/s) are displayed as a function of the imposed temperatures when distilled water is used as feed. A non-linear increase in the flux, as shown in Fig. 2, with increasing temperature reflects the exponential increase in the vapor pressure which provides the driving force. Figure 2 also shows that the permeate flux increases when the recirculation rate is increased. The effect of a higher recirculation rate is to increase the heat transfer coefficient and thus reduce the effect of temperature polarization. This means that the temperatures at the membrane surface more closely approximate that of the bulk streams, and thus the transmembrane temperature difference is greater. This produces a greater driving force and consequently enhances the flux. The figure shows that the present model is capable of predicting permeate fluxes with a reasonable accuracy.

The phenomenon of the temperature polarization causes the temperatures at the membrane surfaces to differ from the bulk temperatures measured in the feed and in the distillate. This phenomenon is present even when the feed is water and causes an important loss in the driving force for transport with regard to the imposed force. The corresponding temperature polarization coefficients (*TPC*) are calculated using the present numerical study and compared with the results obtained by Diez and Vazquez-Gonzalez [18]. These results are shown in Fig. 3 which indicates that the present model is capable of predicting *TPC* with a reasonable accuracy. The results shown in Fig. 3 suggest that the *TPC* is an important factor affecting the mass flux as *TPC* values range between 0.4 and 0.6. Moreover, in the temperature interval studied (T_{bh} varied from

20°C to 50°C), *TPC* varied significantly, decreasing as the temperature increases. This is due to the exponential rise of the vapor pressure curve which makes the permeate flux increases substantially as the temperature rises. These larger mass fluxes involve more important heat fluxes through the liquid phases, increasing the temperature gradient in the liquid boundary layers and so the temperature polarization. For this reason J does not increase with the temperature as fast as the vapor pressure does.

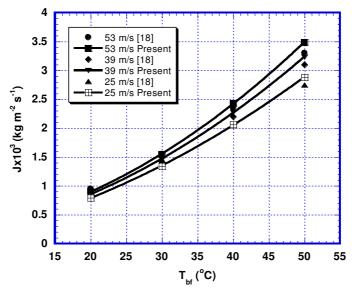


Fig. 2 Water flux vs. feed temperature for feed/permeate inlet velocity of 25, 39 and 53 m/s. For both the present study and the experiment $\Delta T_b = 10^{\circ}$ C.

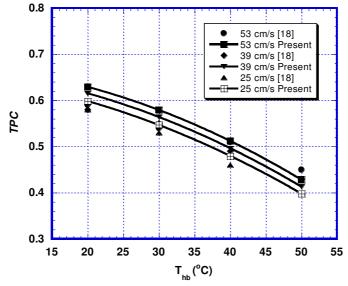


Fig. 3 Temperature polarization coefficient vs. feed temperature for feed/permeate inlet velocity of 25, 39 and 53 m/s. For both the present study and the experiment $\Delta T_b = 10^{\circ}$ C.

4.2 Model Predictions

In the present study, series of simulations were carried out to examine the predictability power of the present model for the DCMD concentration processes of cane-sugar syrup. Numerical simulations are performed to study the effects of varying the controlling parameters such as the cane-sugar syrup feed temperature and concentration on the performance of the DCMD processes. The analysis is made for the inlet temperature of the feed solution ($T_{\rm hi}$) in the range of 55–75°C at 5°C increments, feed solution inlet cane-sugar syrup concentrations of 10–20°Brix at 5°Brix increments, feed solution flow rates ($V_{\rm hi}$) of 1000–2500 ml/min at 500 ml/min increments. The cold solution flow rate ($V_{\rm ci}$) of 500 ml/min, and inlet temperature ($T_{\rm ci}$) of 25°C are kept constants through this study. The membrane employed in the present study is the same as the one used in the validation step.

4.2.1 Velocity, temperature and concentration profiles

In order to demonstrate that the developed model is capable of predicting physically realistic distributions of the quantities of interest, namely velocities, temperature and concentration, simulation runs were performed. The flow, temperature, and concentration profiles results were obtained for temperature difference of 50°C, feed concentration of 20°Brix, and feed flow rate V_h of 1000 ml/min. The feed flow rate V_h is calculated as the feed velocity multiplied by the cell area perpendicular to the liquid flow direction. Examples of the computed results for fields of streamwise velocity, temperature, and concentration are shown in Figs. 4-7.

For the case under study, velocity profiles in the hot solution at different values of x/L are shown in Fig. 4 at x/L = 0, 0.5, and 0.9. The figure shows the boundary layers growth along the channel, with the boundary layer thickness near the membrane wall decreasing from x/L = 0 to 0.9. This could be attributed to the mass permeation from the hot channel to the cold channel through the membrane as a result of temperature difference.

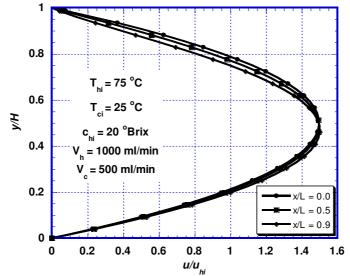


Fig. 4 Streamwise velocity profiles at different axial position along hot channel.

Similarly, the local temperature profiles T (x = constant, y) along the ycoordinate will vary remarkably from x/L = 0.1 to 0.9 as shown in Fig. 5. Figure 5 shows an example of the temperature profiles obtained numerically in the present study at different axial positions in the feed channel. Also the profiles in the hot and cold channels are very similar (not shown). The plotted values are referred to the corresponding local temperature on the membrane hot surface (T_m). When this figure is observed, important temperature drops in the thermal boundary layers can be seen. Taking into account that in this case the whole bulk temperature drop through the hot channel, was about 20°C (as shown in Fig. 6), temperature drops in each thermal boundary layers ranging from 1.6 (corresponding to the axial location of x/L = 0.1) to 4.8° C (for the axial location of x/L = 0.9) are indicative of important temperature polarization phenomena in the system studied.

Figure 6 shows the dependence of the feed and cold solutions temperature at the surface of the membrane on the distance from the channel inlet. The figure also shows the axial distribution of the bulk temperatures in the feed and cold channels for the case considered (concentration of 20° Brix, temperature difference of 50° C and hot feed flow rate of 1000 ml/min). The results shown in the figure reflect the characteristic temperature distribution of a countercurrent flow heat exchanger. In a relatively short entrance region, the transmembrane water flux is very high due to the large driving force. This high flux leads to steep drop of the surface temperature, as it can be seen in the figure. Results obtained in the present study show that the temperature gradients are built along x coordinate in the flow direction inside both the feed and permeate channels.

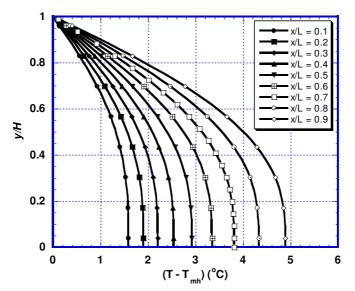


Fig. 5 Temperature profiles at different axial position along the hot channel

Figure 7 shows an example of the concentration polarization distribution obtained numerically in the present study at different axial positions in the feed channel for the case considered (concentration of 20° Brix, temperature difference of 50° C and hot feed flow rate of 1000 ml/min). It is obvious from the figure that the hot

solution concentration is increased as flow goes downstream in the channel due to the pure water permeation through the membrane as a result of temperature difference between the hot and cold streams. The concentration at the vicinity of the membrane wall increases as the flow moves downstream suggesting that the species convection towards the wall is much larger than the species diffusion away from the wall. This concentration build up near the membrane wall will adversely affect the driving force of membrane distillation as can be seen later.

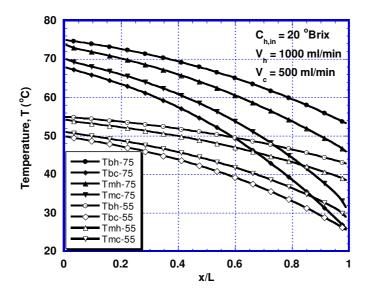


Fig. 6 Axial distributions of hot and cold streams bulk/wall temperatures.

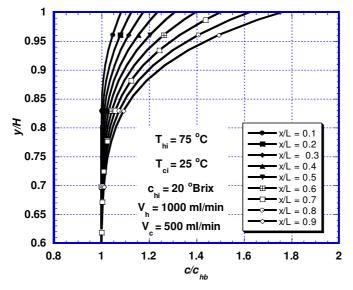


Fig. 7 Concentration polarization profiles at different axial position along hot channel

4.2.2 Effect of feed temperature and concentration on the DCMD

The operating parameters that impact DCMD performance are feed and permeate temperatures and flow rates (which impact the heat transfer coefficients). For example, performance (evaluated in terms of the flux) improves monotonically as the feed temperature is increased from its lower limit of 0°C to its upper limit of 100°C (limits for MD of dilute aqueous solutions) [19]. Feed and permeate flow rates exhibit similar influence [19].

Computer simulations used to investigate the influence of ΔT , the difference between feed and permeate temperatures, for a specific case of DCMD performance are reported here. Figure 8 shows the distillate fluxes as a function of the average temperature differences when different feed cane-sugar syrup concentrations in the membrane module are considered. Figure 8 shows that the permeation flux at different values of ΔT at constant permeate temperature of 25°C is increased as ΔT is increased. A non-linear increase in the flux with increasing temperature reflects the exponential increase in the vapor pressure which provides the driving force. Figure 8 also shows that the flux decreases in an approximately linear way with the feed concentration in the studied range. This can be attributed to the fact that the permeation driving force (vapor pressure) is increased as ΔT is increased and decreased as the feed concentration is increased. These results agree very well with the experimental results of Nene *et al.* [20].

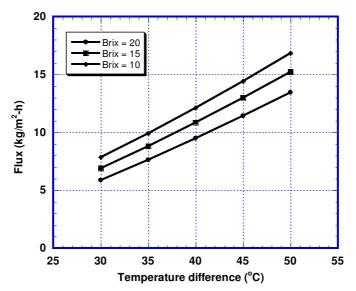


Fig. 8 Membrane distillation of cane-sugar syrup for various values of ΔT (at constant cold side temperature of 25°C)

4.2.3 Effect of feed flow rate on the DCMD

The variation of permeate flux for different values of feed flow rate V_h is shown in Fig. 9. It has been noted that as the feed flow rate increase, the permeate flux increases. The permeate fluxes increase with feed flow rate, and they seem to reach maximum values

asymptotically for higher feed flow rate. Figure 9 shows that the permeate flux increases with the recirculation rate. This can be attributed to the fact that increasing the recirculation rate is to increase the heat transfer coefficient and thus reduce the effect of temperature polarization (*TPC* is increased with V_h). This means that the temperatures at the membrane surface more closely approximate that of the bulk streams, and thus the trans-membrane temperature difference is greater. This produces a greater driving force and consequently enhances the flux.

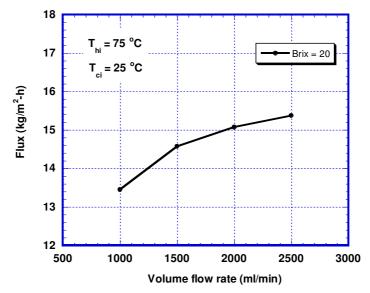


Fig. 9 Membrane distillation of cane-sugar syrup for various feed flow rates.

4.2.4 Temperature and concentration polarization coefficients

A commonly used measure of the magnitudes of the boundary layer heat transfer resistances relative to the total heat transfer resistance of the system is given by the temperature polarization coefficient: *TPC* given by Eq. (21). The value of *TPC* approaches unity for well designed systems, and it approaches zero for systems that are hampered by large boundary layer resistances (high degrees of temperature and concentration polarization) [18]. The highest values [18] of *TPC* reported in the literature for pure water DCMD fall within a range of 0.4 (high fluxes) to 0.7 (low fluxes). Figure 10 shows the calculated values of *TPC* for the cane-sugar syrup DCMD results described above. At very low T_{hi} (low flux), *TPC* ranges from 0.5264 for the 20°Brix feed to 0.4496 for the 10°Brix feed. Even at higher T_{hi} (high flux), the value of *TPC* still lies above the 0.35 value.

As indicated previously after evaluating *TPC* we have evaluated the concentration polarization from the Eq. (21). It was calculated that the membrane wall concentration was as far as 8% higher than the bulk concentration ($c_m/c_{bh}=1.04$) as shown in Fig. 7. In the present study, this concentration increase means a decrease of the vapor pressure to about 0.3%. As a consequence of the concentration polarization leads to a negligible reduction in the permeate flux in the studied cases. This is an important result as concentration polarization is a major cause of flux reduction in

other membrane processes such as ultrafiltration. In the present study, *CPC* is increased linearly with feed temperatures as shown in Fig. 11.

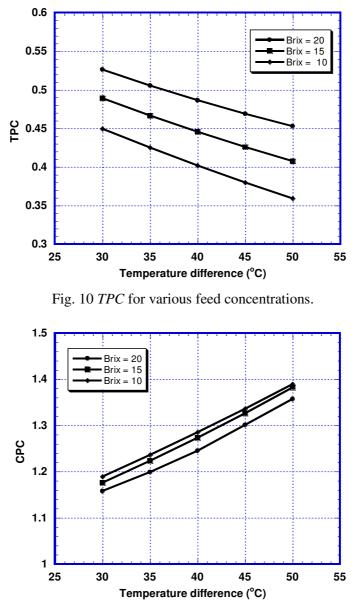


Fig. 11 CPC for various feed concentrations.

4.2.5 Thermal efficiency

One of the primary concerns in DCMD, with regard to effective energy consumption, is limiting the amount of heat lost in the process. The amount of heat that is lost by conduction through the membrane and the membrane module is a function of the

module design and the thermal conductivities of the module material and the membrane.

In order to estimate the heat lost, calculations were performed in the following way. The latent heat of evaporation is the heat used effectively and is obtained from the permeate flux. The increase in heat of the cooling water is a sum of the latent heat of evaporation and the conduction heat lost through the membrane from the feed to cooling water. Figure 12 shows the effect of the hot and cold solution inlet temperatures on the thermal efficiency. The thermal efficiency increases by about 11, 12, and 13% as $\Delta T_{\rm hi}$ increases from 30 to 50°C for feed concentration of 10, 15, and 20°Brix; respectively. Increasing the inlet temperature of the hot solution thus does not only increase the permeate flux but also improves the thermal efficiency.

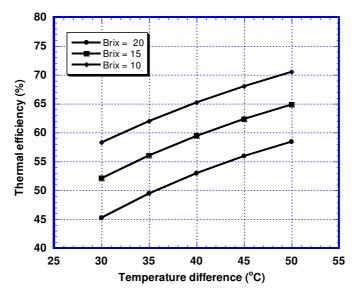


Fig. 12 Thermal efficiency for various feed concentrations.

5. CONCLUSIONS

DCMD process has been numerically investigated. This was facilitated by solving the elliptic type of the governing equations using the SIMPLE pressure-correction algorithm for pressure field in connection with the high order QUICK scheme. The alternating direction implicit ADI scheme, which makes use of the TDMA in solving the resulting coefficient matrix, was used to solve the governing equations to reduce the number of iterations. A scheme to secure the overall mass conservation was also employed.

The results obtained in the present study (e.g. Fig. 9) indicate that it was possible to consistently remove water from the cane sugar solution at a steady state value approximating 15.0 kg/m²/h. The viscosity of the cane sugar solution at high concentration induces high polarization phenomena. The proposed method succeeds in predicting the effects of the controlling parameters on the DCMD performance. The following points can be drawn from the numerical simulation:

- (1) Difference of temperatures between hot and cold solutions is important factor for the capacity of direct contact membrane distillation process. The distillate flux increases with increasing the temperature difference,
- (2) The distillate flux decreases as the feed initial concentration increases,
- (3) The distillate flux increases with the flow rate through the feed channel,
- (4) The process thermal efficiency enhanced as the feed temperature is increased.
- (5) The influence of temperature polarization on the effectiveness of DCMD in cane sugar solution concentration has been found to be important.

The results were compared with the available data and the agreement is satisfactory. This study can be used for DCMD process improvement and optimization.

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نمذجة عملية تركيز عصير قصب السكر الخام باستخدام التقطير بأغشية الاتصال المدجة عملية تركيز عصير قصب المباشر (DCMD) على كامل عبد الرحمن قسم الهندسة الميكانيكية – كلية الهندسة

التقطير بالأغشية هي تقنية واعدة لإجراء عمليات الفصل التي كانت تتم عادة باستخدام عمليات الفصل التقليدية مثل التقطير الحراري والتناضح العكسي. القوة الدافعة للتقطير بالأغشية هي عبارة عن الفرق في ضغط البخار الذي يمكن الحصول عليه من خلال وجود محلول عند درجة حرارة أقل من درجة حرارة محلول التغذية بحيث يتلامس مباشرة مع جانب النفاذ للغشاء مما يهيئ النمط المعروف باسم التقطير بأغشية الاتصال المباشر (DCMD). هذه العملية تتواءم بمثالية مع عمليات تركيز المحاليل المخففة مثل عصائر الفاكهة والمحاليل السكرية.

تم إجراء الدراسة العددية الحالية من اجل شرح كيفية تطبيق تقنية التقطير بأغشية الاتصال المباشر (DCMD) في عملية تركيز عصير قصب السكر. الهدف الرئيسي من هذه الدراسة هو توفير تحليل عددي تفصيلي لعمليات انتقال الحرارة والكتلة في تقنية التقطير بأغشية الاتصال المباشر (DCMD) وتوفير معلومات تفصيلية أساسية مفيدة عن طبيعة العملية والتي يحتاج إليها في تحسين وتوفيق العملية. في هذا الصدد، فإن الدراسة الحالية أجريت من اجل استكشاف تأثيرات العوامل الحاكمة مثل درجة حرارة وتركيز محلول التغذية وسرعات المحاليل الساخنة والباردة على كمية سريان السائل المقطر.

الطريقة المطورة تسمح بالحل العددي لمعادلات الحفظ لانتقال كمية الحركة ، الطاقة والكتلة مع الأخذ في الاعتبار للنفاذة خلال الغشاء. توزيعات السرعات ودرجات الحرارة داخل قنوات محلول التغذية والمحلول البارد أمكن الحصول عليها بالإضافة إلى توزيعات تركيز عصير قصب السكر في قناة التغذية. بعض الاستنتاجات التي أمكن الحصول عليها من هذه الدراسة تتمثل في الآتي: (1) كمية سريان السائل المقطر تزداد مع زيادة درجة حرارة محلول التغذية، (2) كمية سريان السائل المقطر تتخفض مع زيادة تركيز محلول التغذية، (3) كمية سريان السائل المقطر تزداد مع زيادة معدلات سريان محلول التغذية. تم مقارنة نتائج هذه الدراسة مع البيانات المتاحة ووجد أن التطابق بينها جيد.